

Vibrations

By the end of this lesson, you should be able to:

- Describe the motion over time of vibrating systems
- Understand the natural frequencies of systems
- Be able to calculate resonant frequencies
- Have a better appreciation for ODEs ☺

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Outline

- Conservative Systems (Mass + Spring)
- Damped Vibrations (Mass + Spring + Resistance)
- Forced Vibrations

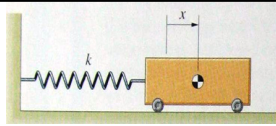
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Vibrations

- Anytime you have a spring and a mass, you get vibrations
- You can think of vibrations as oscillations between kinetic (mass) and potential (spring) energy
- These vibrations may be damped by resistance
- We use Differential Equations to solve for the motion of vibrating systems over time
- We can then use them to tune the values of stiffness, mass, and resistance to amplify (ie, harmonics) or attenuate vibrations

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Conservative Systems

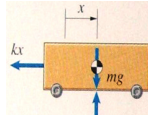


1) Draw Free-Body Diagram

2) Sum Forces $\Sigma F_x = -kx = ma$

3) Change into ODE $-kx = m \frac{d^2x}{dt^2}$

$$\frac{d^2x}{dt^2} + \frac{k}{m}x = 0$$



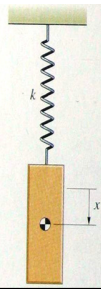
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Another Conservative System

2) Sum Forces $\Sigma F_y = mg - ky = ma$

3) Change into ODE $mg - ky = m \frac{d^2y}{dt^2}$

$$\frac{d^2y}{dt^2} + \frac{k}{m}y = g$$



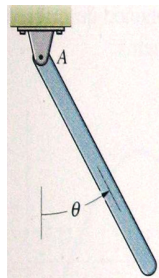
A third Conservative System

2) Sum Moments $\Sigma M_A = -mg \sin \theta = I \alpha$ $I_A = \frac{1}{12}ml^2 + m\left(\frac{l}{2}\right)^2 = \frac{ml^2}{3}$

3) Change into ODE $-mg \frac{l}{2} \sin \theta = \frac{ml^2}{3} \frac{d^2\theta}{dt^2}$

$$\frac{d^2\theta}{dt^2} + \frac{3g}{2l} \sin \theta = 0$$

3) For small θ $\frac{d^2\theta}{dt^2} + \frac{3g}{2l} \theta = 0$



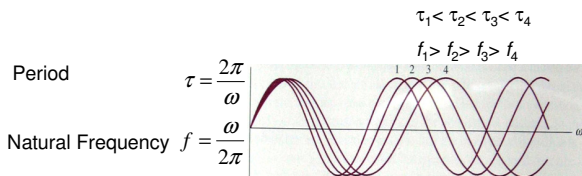
Solutions to Homogeneous ODEs

$$\frac{d^2x}{dt^2} + \omega^2 = 0$$

In the examples above:
 $\omega^2 = \frac{k}{m}, \frac{3g}{2l}$

$$x = A \sin \omega t + B \cos \omega t = E \sin(\omega t - \phi)$$

$$A = E \cos \phi, \quad B = -E \sin \phi$$

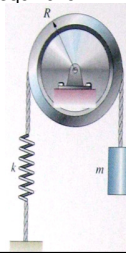


Example

The pulley has radius R and moment of inertia I , and the cable does not slip relative to the pulley. The mass m is displaced downward a distance h from its equilibrium position and released from rest at $t=0$.

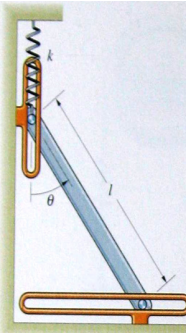
What is the natural frequency of the resulting vibrations?

Determine the position of the mass relative to its equilibrium position as a function of time



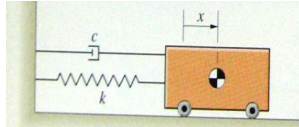
Example

The spring attached to the slender bar of mass m is unstretched when $\theta=0$. Neglecting friction, determine the natural frequency of small vibrations of the bar relative to its equilibrium position



Damped Motion

- All real motion encounters resistance
- Resistance is good! It prevents vibrations
- As engineers, we often want to find the "optimum" resistance – just enough to prevent vibrations, without slowing us down too much.
- This optimal damping is called *Critical Damping*.
Not enough is called *Subcritical Damping*.
Too much is called *Supercritical Damping*.



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Damped Vibrations

$$-c \frac{dx}{dt} - kx = m \frac{d^2x}{dt^2}$$

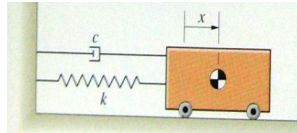
$$\omega^2 = \frac{k}{m}, \quad \zeta = \frac{c}{2m}$$

$$\frac{d^2x}{dt^2} + 2\zeta \frac{dx}{dt} + \omega^2 x = 0$$

Subcritical Damping: $\zeta < \omega$

Critical Damping: $\zeta = \omega$

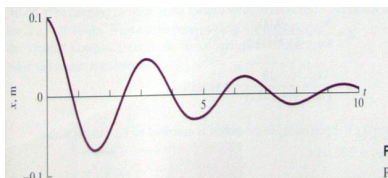
Supercritical Damping: $\zeta > \omega$



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Damped Vibrations

- Amplitude Decreases over time
- Frequency is slower than undamped: $\omega_d = \sqrt{\omega^2 - d^2}$
- Frequency remains constant – it doesn't get slower over time; only amplitude changes over time



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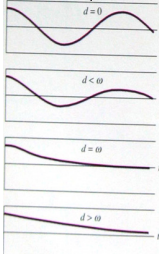
Damped Vibrations

$$\frac{d^2x}{dt^2} + 2\zeta \frac{dx}{dt} + \omega^2 x = 0$$

Subcritical Damping: $\zeta < \omega$ $x = e^{-\zeta t} (A \sin \omega_d t + B \cos \omega_d t)$

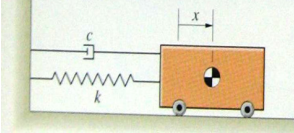
Supercritical Damping: $\zeta > \omega$ $x = C e^{-(\zeta-h)t} + D e^{-(\zeta+h)t}$ $h = \sqrt{\zeta^2 - \omega^2}$

Critical Damping: $\zeta = \omega$ $x = C e^{-\zeta t} + D t e^{-\zeta t}$ $(h=0)$



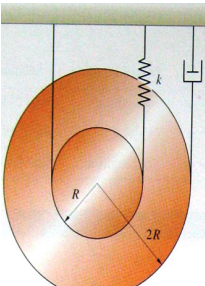
Example

The damped spring-mass oscillator has mass $m = 2 \text{ kg}$, spring constant $k = 8 \text{ N/m}$, and damping constant $c = 1 \text{ N-s/m}$. At $t = 0$, the mass is released from rest in the position $x = 0.1 \text{ m}$. Determine its position as a function of time.



Example

The 40 N stepped disk is released from rest with the spring unstretched. Determine the position of the center of the disk as a function of time if $R = 1 \text{ m}$, $k = 10 \text{ N/m}$, $c = 4 \text{ N-s/m}$, and the moment of inertia expressed in terms of the mass m of the disk is $I = 3mR^2$.



Forced Vibrations

A motor, or a person, can vibrate at a given frequency. These vibrations interact with the natural vibrations of the system. If they happen to be at the same frequency, resonance can occur, amplifying the vibrations.

$$\frac{d^2x}{dt^2} + 2\zeta\frac{dx}{dt} + \omega^2x = a(t)$$

Mathematically, forced vibrations do not produce homogeneous solutions – a particular solution must be added to the homogeneous solution: $x = x_h + x_p$

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Sinusoidal Forces

$$\frac{d^2x}{dt^2} + 2\zeta\frac{dx}{dt} + \omega^2x = a(t) \quad a(t) = a_0 \sin \omega_0 t + b_0 \cos \omega_0 t$$

$$x_p = A_p \sin \omega_0 t + B_p \cos \omega_0 t$$

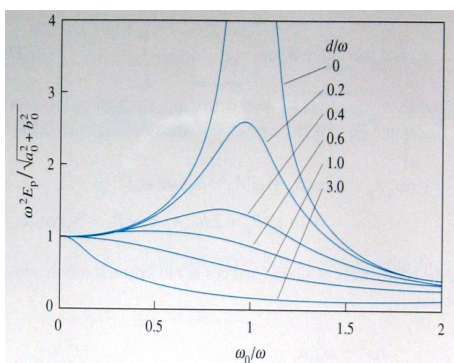
$$A_p = \frac{(\omega^2 - \omega_0^2)a_0 + 2\zeta\omega_0 b_0}{(\omega^2 - \omega_0^2)^2 + 4\zeta^2\omega_0^2}$$

$$B_p = \frac{(\omega^2 - \omega_0^2)b_0 - 2\zeta\omega_0 a_0}{(\omega^2 - \omega_0^2)^2 + 4\zeta^2\omega_0^2}$$

The system approaches the steady-state solution over time as the natural system dampens out

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Resonance

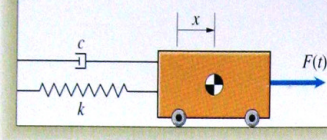


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Example

An engineer designing a vibration isolation system for an instrument console models the console and isolation system by a damped spring-mass oscillator with mass $m = 2$ kg, spring constant $k = 8$ N/m, and damping constant $c = 1$ N·s/m. To determine the system's response to external vibration, she assumes that the mass is initially stationary with the spring unstretched, and at $t = 0$ a force $F(t) = 20 \sin 4t$ N is applied to the mass.

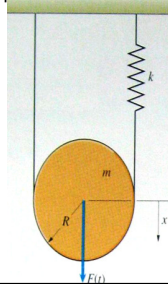
- a) What is the amplitude of the steady-state solution?
- b) What is the position of the mass as a function of time?



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Example

The homogeneous disk has radius $R = 2$ ft and mass $m = 4$ slugs. The spring constant is $k = 30$ lb/ft. The disk is initially stationary in its equilibrium position, and at $t = 0$ a downward force $F(t) = 12 + 12t - 0.6t^2$ lb is applied to the center of the disk. Determine the position of the center of the disk as a function of time.



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